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# RADIAL TEMPERATURE DISTRIBUTION IN ISOTACHOPHORESIS COLUMNS OF CIRCULAR CROSS-SECTION

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# SUMMARY

An analysis of the unsteady-state heat-conduction problem in isotachophoretic columns of circular cross-section is given, based on a linear variation of electrical conductivity with temperature and taking into account the effects of a finite wall thickness. It is seen that previous analyses, based on constant electrical conductivity, tend to underpredict actual temperature distributions considerably. The use of thick-walled columns is seen to be advantageous both from the viewpoint of higher resolution and of increased production.

#### INTRODUCTION

Joule heating, caused by the passage of current through an electrophoresis column, produces radial temperature gradients in which the temperature is highest at the axis of the column and declines toward the periphery. A precise determination of radial temperature distributions is essential for two very important reasons. First, because ionic mobilities are temperature dependent (increasing about  $2\%/^{\circ}$ K), a radial temperature distribution produces variations in migration velocity across the tube, thus causing "bowing" of the frontal (ionic species interface) regions. Secondly, in the case of living cells relatively small temperature changes can be tolerated over the cross-section of the column if cell damage is to be avoided (0 to  $37^{\circ}$ C when using an aqueous solvent). This requires the solution to the transient heat-conduction problem as the system must be designed in such a way that the time required for separation is less than the time required for the system to reach this critical temperature at which cell destruction begins.

We consider the isotachophoresis to occur in a glass-walled column of circular cross-section. The voltage drop along the tube is assumed to be independent of the temperature and hence of the radial location. It follows that the current density is then a function of radial location because of the temperature dependence of the electrical conductivity.

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The present solution is based on the following assumptions.

(1) Convective effects may be ignored;

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(2) The fluid supporting medium is essentially at rest with the various ionic species drifting through it in the axial direction;

(3) The thermal properties (density, specific heat and thermal conductivity) are independent of temperature;

(4) The electrical conductivity is a linear function of the temperature over the range of temperatures of interest;

(5) The temperature distribution may be obtained as the solution of a conduction problem involving only the radial coordinate and time.

Some further comments are in order with respect to these-assumptions.

Assumption 5 will be a good approximation in any longitudinal (axial) region sufficiently far away from any fronts where it is obvious that strong longitudinal temperature gradients must exist. It may be argued that it is precisely in the region of the fronts that we wish to determine the effects of non-uniformities in the temperature distribution. However, the present approach is a legitimate first step towards the more general problem and certainly should permit a first-order estimate of the temperature dependence of the species migration-velocity profile to be determined. Furthermore, it is clear that there is always one region (the terminator) where the heat generation is a maximum and this region is not a sample region. Thus, if the maximum design temperature is specified in this region, there will be no danger of the sample species becoming overheated. Present calculations permit the determination of the time taken to reach maximum design temperature and the corresponding maximum electric field strength.

Assumption 1 will certainly be valid in gels, and because we are considering applications for manufacturing in space where operation will occur in a gravity-free environment, it should also be valid in free solution provided electroconvective effects may be neglected. Preliminary investigations in Skylab indicate that this may not be a major factor.

In view of assumption 2 the present results will not apply to any system using a free solution with counterflow.

Assumption 3 has been checked by comparing the "exact" steady-state constant-properties solution with the results of a steady-state perturbation analysis which permits variation in the thermal properties with temperature. It appears that the effect of neglecting thermal-property variations is very small and that the results of a constant-property analysis are conservative. Details of this particular point are discussed in the Appendix.

# PREVIOUS WORK

Martin and Everaerts<sup>1</sup> have considered a steady-state analysis under the assumptions of constant thermal and electrical conductivity and specific heat. They obtain an expression for the difference between the temperature at the center of the column and that at any radius r. This temperature distribution is parabolic and neglects the effects of column wall thickness. Assuming a mobility increase with temperature of  $2\%/^{\circ}K$ , they obtain an expression for the variation of mobility due to this radial temperature distribution.

Konstantinov and Oshurkova<sup>2</sup>, using the same assumptions as above, obtain a similar expression for the difference in temperature between a point at radius r and the inside column wall. Using the  $2\%/^{\circ}K$  mobility variation with temperature they arrive at an expression for the velocity variation across the column.

Hjertén<sup>3</sup>, considering free zone electrophoresis, makes the same assumptions as in the above two cases, and also imposes the condition that the temperature variation across the column be much smaller than the cooling-bath temperature and thus obtains a parabolic velocity profile. He states that the migration-velocity profile in a temperature gradient is determined almost entirely by the temperature variation of viscosity, and suggests the use of a counterflow of buffer to flatten curved zones.

Routs<sup>4</sup>, basing his observations on the work of Hjertén, notes that, because electrical conductivity decreases from the leading electrolyte to the terminator, the boundaries of the zones near the terminator will be more curved than the boundaries of the high-conductivity zones. He further states that the only way to straighten the fronts is to reduce the field strength because the use of counterflow to restore the boundaries of the terminating zones includes the danger of destroying the straighter fronts of the leading ion zones.

Brown and Hinckley<sup>5</sup> consider a linear variation of electrical conductivity with temperature and include the effects of the presence of a finite wall thickness and thus obtain expressions for the temperature profiles in the electrolyte solution and in the column wall for the steady-state condition, based on a constant-temperature boundary condition on the outside wall.

# STATEMENT OF THE PROBLEM

Mathematically the problem may be stated as follows:

$$K_{1}\left\{\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial T_{1}}{\partial r}\right)\right\} = \frac{\partial T_{1}}{\partial t} - \frac{Q_{e}}{Q_{1} c_{1}}, \quad 0 \leq r \leq R_{1}$$

$$K_{2}\left\{\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial T_{2}}{\partial r}\right)\right\} = \frac{\partial T_{2}}{\partial t}, \quad R_{1} \leq r \leq R_{2}$$

$$T_{1} = T_{2} \text{ at } r = R_{1}, \quad t > 0$$

$$k_{1} \frac{\partial T_{1}}{\partial r} = k_{2} \frac{\partial T_{2}}{\partial r} \text{ at } r = R_{1}, \quad t > 0$$

$$T_{1} \text{ finite at } r = 0.$$

$$k_{2} \frac{\partial T_{2}}{\partial r} + h \left(T_{2} - T_{\infty}\right) = 0 \text{ at } r = R_{2}, \quad t > 0$$

$$T_{1} = T_{\infty} \text{ in } 0 \leq r \leq R_{1}, \quad t = 0$$

 $T_2 = T_\infty$  in  $R_1 \leq r \leq R_2, t = 0.$ 

We introduce the dimensionless variables

$$\theta = \frac{T - T_{\infty}}{T_{\infty}}, \, \xi = \frac{r}{R_2}, \, \tau = \frac{K_1 t}{R_2^2}$$

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and assume that

$$Q_{e} = Q_{0} (1 + \alpha \theta) = JE^{2}\sigma = JE^{2}\sigma_{0} (1 + \alpha \theta)$$

where the heat generation per unit volume and the electrical conductivity are denoted by  $Q_0$  and  $\sigma_0$ , respectively, when evaluated at  $\theta = 0$ . We also denote

$$S = \frac{Q_0 R_2^2}{k_1 T_\infty}$$
$$\gamma = \frac{h R_2}{k_2}$$
$$\xi_1 = R_1/R_2$$

Then the statement of the problem in dimensionless variables is:

$$\frac{1}{\xi} \frac{\partial}{\partial \xi} \left( \xi \frac{\partial \theta_1}{\partial \xi} \right) = \frac{\partial \theta_1}{\partial \tau} - S \left( 1 + \alpha \theta_1 \right), \quad 0 \leqslant \xi \leqslant \xi_1$$

$$\frac{1}{\xi} \frac{\partial}{\partial \xi} \left( \xi \frac{\partial \theta_2}{\partial \xi} \right) = \frac{K_1}{K_2} \frac{\partial \theta_2}{\partial \tau}, \quad \xi_1 \leqslant \xi \leqslant 1$$

$$\theta_1 = \theta_2 \text{ at } \xi = \xi_1$$

$$k_1 \frac{\partial \theta_1}{\partial \xi} = k_2 \frac{\partial \theta_2}{\partial \xi} \text{ at } \xi = \xi_1$$

$$\theta_1 \text{ finite at } \xi = 0$$

$$\frac{\partial \theta_2}{\partial \xi} + \gamma \theta_2 = 0 \text{ at } \xi = 1$$

$$\theta_1 = 0 \text{ in } 0 \leqslant \xi \leqslant \xi_1, \quad \tau = 0$$

$$\theta_2 = 0 \text{ in } \xi_1 \leqslant \xi \leqslant 1, \quad \tau = 0$$

This problem is split into steady-state and transient parts by writing

$$\theta(\xi,\tau) = \theta^{s}(\xi) + \theta^{\iota}(\xi,\tau)$$

Steady-state problem

$$\frac{1}{\xi} \frac{\mathrm{d}}{\mathrm{d}\xi} \left(\xi \frac{\mathrm{d}\theta_1^{\mathrm{s}}}{\mathrm{d}\xi}\right) = -S\left(1 + \alpha \theta_1^{\mathrm{s}}\right), \ 0 \leqslant \xi \leqslant \xi_1$$
$$\frac{1}{\xi} \frac{\mathrm{d}}{\mathrm{d}\xi} \left(\xi \frac{\mathrm{d}\theta_2^{\mathrm{s}}}{\mathrm{d}\xi}\right) = 0, \ \xi_1 \leqslant \xi \leqslant 1$$
$$\theta_1^{\mathrm{s}} = \theta_2^{\mathrm{s}} \text{ at } \xi = \xi_1; \ \theta_1^{\mathrm{s}} \text{ finite at } \xi = 0$$

$$k_1 \frac{\mathrm{d} \theta_1^{\mathrm{s}}}{\mathrm{d} \xi} = k_2 \frac{\mathrm{d} \theta_2^{\mathrm{s}}}{\mathrm{d} \xi} \text{ at } \xi = \xi_1$$

$$\frac{\mathrm{d}\theta_2^{\mathrm{s}}}{\mathrm{d}\xi} + \gamma \theta_2^{\mathrm{s}} = 0 \text{ at } \xi = 1$$

The solutions are

$$\theta_1^{s} = A_1 J_0 \left(\beta \xi\right) - \frac{1}{\alpha}, \ \beta^2 = \alpha S$$
$$\theta_2^{s} = A_2 \left[\ln \xi - \frac{1}{\gamma}\right]$$

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where

$$A_{1} = \frac{1}{\alpha \left[ J_{0} \left(\beta \xi_{1}\right) + \frac{k_{1}}{k_{2}} \beta \xi_{1} J_{1} \left(\beta \xi_{1}\right) \left\{ \ln \xi_{1} - \frac{1}{\gamma} \right\} \right]} - \frac{k_{1}}{k_{2}} \beta \xi_{1} J_{1} \left(\beta \xi_{1}\right)}$$
$$A_{2} = \frac{-\frac{k_{1}}{k_{2}} \beta \xi_{1} J_{1} \left(\beta \xi_{1}\right)}{\alpha \left[ J_{0} \left(\beta \xi_{1}\right) + \frac{k_{1}}{k_{2}} \beta \xi_{1} J_{1} \left(\beta \xi_{1}\right) \left\{ \ln \xi_{1} - \frac{1}{\gamma} \right\} \right]}$$

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Transient problem

$$\frac{1}{\xi} \frac{\partial}{\partial \xi} \left( \xi \frac{\partial 0_1^i}{\partial \xi} \right) = \frac{\partial 0_1^i}{\partial \tau} - \alpha S \theta_1^i, \ 0 \le \xi \le \xi_1$$

$$\frac{1}{\xi} \frac{\partial}{\partial \xi} \left( \xi \frac{\partial 0_2^i}{\partial \xi} \right) = \frac{K_1}{K_2} \frac{\partial \theta_2^i}{\partial \tau}, \ \xi_1 \le \xi \le 1$$

$$\theta_1^i \text{ finite at } \xi = 0$$

$$\theta_1^i = \theta_2^i \text{ at } \xi = \xi_1, \ \tau > 0$$

$$k_1 \frac{\partial \theta_1^i}{\partial \xi} = k_2 \frac{\partial \theta_2^i}{\partial \xi} \text{ at } \xi = \xi_1, \ \tau > 0$$

$$\frac{\partial \theta_2^i}{\partial \xi} + \gamma \theta_2^i = 0 \text{ at } \xi = 1, \ \tau > 0$$

$$\theta_1^i = -\theta_1^s \text{ in } 0 \le \xi \le \xi_1, \ \tau = 0$$

$$\theta_2^i = -\theta_2^s \text{ in } \xi_1 \le \xi \le 1, \ \tau = 0$$
This problem is solved by the method of Tittle<sup>6</sup>.

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We seek a solution of the form

$$\theta_j^1(\xi,\tau) = \sum_{n=1}^{\infty} A_n X_{jn} e^{-\alpha_j \beta_{jn}^2 \tau} \text{ in region } j; j = 1,2.$$

It is easily shown that for region 1 we must have

 $X_{1n}(\xi) = J_0(\lambda_{1n}\xi)$ 

where  $\lambda_{1_n}^2 = \beta^2 + \beta_{1_n}^2$  and  $\beta_{1_n}$  are the eigenvalues and  $\alpha_1 = 1$ . Similarly, for region 2 we obtain

$$X_{2n} = B_n \left[ J_0 \left( \beta_{2n} \, \xi \right) + C_n \, Y_0 \left( \beta_{2n} \, \xi \right) \right]$$

and  $\alpha_2 = K_2/K_1$ . The eigenvalues  $\beta_{2n}$  are related to  $\beta_{1n}$  by

$$\beta_{1n}^2 = \alpha_2 \beta_{2n}^2$$

The boundary condition at  $\xi = 1$  leads to

$$C_{n} = \frac{\gamma J_{0}(\beta_{2n}) - \beta_{2n} J_{1}(\beta_{2n})}{\beta_{2n} Y_{1}(\beta_{2n}) - \gamma Y_{0}(\beta_{2n})}$$

Then equating  $\theta_1^t$  and  $\theta_2^t$  at  $\xi = \xi_1$  gives

$$B_n = \frac{J_0(\lambda_{1n} \xi_1)}{[J_0(\beta_{2n} \xi_1) + C_n Y_0(\beta_{2n} \xi_1)]}$$

Finally, equating  $k_1 \frac{\partial \theta_1^t}{\partial \xi} = k_2 \frac{\partial \theta_2^t}{\partial \xi}$  at  $\xi = \xi_1$  gives

$$k_1 \lambda_{1n} J_1 (\lambda_{1n} \xi_1) = k_2 \beta_{2n} B_n [J_1 (\beta_{2n} \xi_1) + C_n Y_1 (\beta_{2n} \xi_1)]$$

as the transcendental equation to be solved for the eigenvalues  $\beta_{1_n}$  (or  $\beta_{2_n}$ ). The coefficients  $\dot{A}_n$  are then given by

$$A_{n} = \frac{\int_{0}^{\xi_{1}} \left[-\theta_{1}^{s}\right] X_{1n} \xi d\xi + \left(\frac{K_{1}}{K_{2}} \frac{k_{2}}{k_{1}}\right) \int_{\xi_{1}}^{1} \left[-\theta_{2}^{s}\right] X_{2n} \xi d\xi}{\int_{0}^{\xi_{1}} X_{1n}^{2} \xi d\xi + \left(\frac{K_{1}}{K_{2}} \frac{k_{2}}{k_{1}}\right) \int_{\xi_{1}}^{1} X_{2n}^{2} \xi d\xi}$$

The various integrations may be performed giving:

$$\int_{0}^{\xi_{1}} \left[-\theta_{1}^{\xi}\right] X_{1n} \xi d\xi = \frac{1}{\alpha} \frac{\xi_{1}}{\lambda_{1n}} J_{1} \left(\lambda_{1n} \xi_{1}\right) - \frac{A_{1} \xi_{1}}{\beta_{1n}^{2}} \left[\lambda_{1n} J_{0} \left(\beta \xi_{1}\right) J_{1} \left(\lambda_{1n} \xi_{1}\right) - \beta J_{0} \left(\lambda_{1n} \xi_{1}\right) J_{1} \left(\beta \xi_{1}\right)\right]$$

$$= \int_{\xi_{1}}^{1} \left[-\theta_{2}^{\xi}\right] X_{2n} \xi d\xi = \frac{A_{2} B_{n}}{\gamma \beta_{2n}} \left[J_{1} \left(\beta_{2n}\right) - \xi_{1} J_{1} \left(\beta_{2n} \xi_{1}\right) + C_{n} \left\{Y_{1} \left(\beta_{2n}\right) - \xi_{1} Y_{1} \left(\beta_{2n} \xi_{1}\right)\right\}\right] - \frac{A_{2} B_{n}}{\beta_{2n}} \left[-\xi_{1} \ln \xi_{1} J_{1} \left(\beta_{2n} \xi_{1}\right) + \frac{1}{\beta_{2n}} \left\{J_{0} \left(\beta_{2n}\right) - J_{0} \left(\beta_{2n} \xi_{1}\right)\right\}\right] - \frac{A_{2} B_{n} C_{n}}{\beta_{2n}} \left[-\xi_{1} \ln \xi_{1} Y_{1} \left(\beta_{2n} \xi_{1}\right) + \frac{1}{\beta_{2n}} \left\{Y_{0} \left(\beta_{2n}\right) - Y_{0} \left(\beta_{2n} \xi_{1}\right)\right\}\right]$$

$$\int_{\xi_{1}}^{\xi_{1}} X_{1n}^{2} \,\xi d\xi = \frac{\xi_{1}^{2}}{2} \left[ J_{1}^{2} (\lambda_{1n} \,\xi_{1}) + J_{0}^{2} (\lambda_{1n} \,\xi_{1}) \right]$$
  
$$\int_{\xi_{1}}^{1} X_{2n}^{2} \,\xi d\xi = B_{n}^{2} \left[ \frac{1}{2} \left\{ J_{1} (\beta_{2n}) + C_{n} \,Y_{1} (\beta_{2n}) \right\}^{2} + \frac{1}{2} \left\{ J_{0} (\beta_{2n}) + C_{n} \,Y_{0} (\beta_{2n}) \right\}^{2} - \frac{\xi_{1}^{2}}{2} \left\{ J_{1} (\beta_{2n} \,\xi_{1}) + C_{n} \,Y_{1} (\beta_{2n} \,\xi_{1}) \right\}^{2} - \frac{\xi_{1}^{2}}{2} \left\{ J_{0} (\beta_{2n} \,\xi_{1}) + C_{n} \,Y_{0} (\beta_{2n} \,\xi_{1}) \right\}^{2} \right]$$

### **RESULTS AND DISCUSSION**

Numerical calculations were performed on the CDC-6400 computer at the University of Arizona Computer Center. Figs. 1–3 show centerline transient temperature response for various heating rates for three different values of the wall-thickness parameter  $\xi_1$ . In each case, the dimensionless heat-transfer coefficient,  $\gamma$ , is equal to infinity (corresponding to a constant outer wall temperature). Table I lists the values of the properties used in making these calculations. For a cooling-bath temperature of 0°C and a maximum allowable temperature in the column of 37°C, we find the maximum allowable value of  $\theta$  to be 0.1355. This maximum allowable value of  $\theta$  is denoted  $\theta_c$ .

An inspection of Figs. 1-3 shows that for a given value of  $\xi_1$ , there will be some value of S below which  $\theta_c$  will never be exceeded and above which  $\theta_c$  will always be



Fig. 1. Centerline transient temperature response for  $\xi_1 = 2/3$ ,  $\gamma = \infty$ . Fig. 2. Centerline transient temperature response for  $\xi_1 = 4/5$ ,  $\gamma = \infty$ .



Fig. 3. Centerline transient temperature response for  $\xi_1 = 9/10$ ,  $\gamma = \infty$ .

exceeded in some finite amount of time. We term this value of S for which  $\theta_c$  is approached asymptotically  $S_{max}$ , and its variation with  $\xi_1$  is shown in Fig. 4.

For the sake of comparison, Fig. 5 shows centerline transient temperature response for a value of the dimensionless heat-transfer coefficient equal to unity. It is clear from this figure that effective cooling is extremely important if one is to use high enough electric field strengths to give acceptable resolution while refraining from "cooking" the sample.

Because the electric field decreases in going from terminator to leader electrolyte zones, S will also decrease from terminator to leader. Fig. 6 depicts the steadystate temperature profile for various values of S. Because ionic mobility is proportional to temperature, this leads to velocity profiles that become progressively less "bowed" from terminator to leader zones.

Fig. 7 shows steady-state temperature distributions for various values of  $\xi_1$  with the heating rate in each case equal to  $S_{max}$ . This figure demonstrates the advantage of thick-walled columns as the total temperature drop within the column

# TABLE I

# DATA USED FOR CALCULATIONS

Parameter	Value
<i>k</i> <sub>1</sub>	$13.9 \cdot 10^{-4} \text{ cal} \cdot \text{cm}^{-1} \cdot \text{sec}^{-1} \cdot ^{\circ}\text{K}^{-1}$
$k_2$	$26.7 \cdot 10^{-4} \text{ cal} \cdot \text{cm}^{-1} \cdot \text{sec}^{-1} \cdot {}^{\circ}\text{K}^{-1}$
$K_1$	$14.779 \cdot 10^{-4} \text{ cm}^2/\text{sec}$
$K_2$	59.333 · 10 <sup>-4</sup> cm <sup>2</sup> /sec
α	7.75



Fig. 4. Variation of maximum heating rate with wall-thickness parameter,  $\gamma = \infty$ . Fig. 5. Centerline transient temperature response for  $\xi_1 = 4/5$ ,  $\gamma = 1$ .



Fig. 6. Steady-state temperature profiles based on a linear variation of electrical conductivity with temperature,  $\xi_1 = 0.8$ ,  $\gamma = \frac{1}{\sqrt{2}}$ 



Fig. 7. Steady-state temperature profiles for  $S = S_{max}$ .

(and hence velocity variation or "bowing" effect) is seen to increase considerably with decreasing wall thickness.

It is interesting to compare the present analysis with the parabolic temperature distributions predicted by previous investigators. The results of Konstantinov and Oshurkova<sup>2</sup> may be written, in terms of our nomenclature, as

$$\theta^{s} = \frac{S}{4} \left( 1 - \frac{\xi^{2}}{\xi_{1}^{2}} \right) \qquad 0 \leq \xi \leq \xi_{1}$$

The above equation is shown plotted in Fig. 8. Comparison with Fig. 6 shows that the parabolic distribution tends to underpredict the temperature, the discrepancy increasing with increasing heating rate. At a value of S equal to 0.4, Fig. 8 predicts that the maximum value of  $\theta$  will be 26.2% less than  $\theta_c$ . From Fig. 6, however, it is seen that at S = 0.4, the maximum value of  $\theta$  is actually 6% greater than  $\theta_c$ . Thus a design based on a parabolic temperature distribution could lead one to believe that a large margin of safety exists with respect to preserving living cells, whereas the actual temperature rise would surely be destructive to such cells.

Because the sharpness of ionic interfaces increases with increasing electric field strength<sup>7</sup>, it may be desirable to operate at values of S greater than  $S_{max}$ . This requires that E and  $R_1$  be chosen in such a way that the time required for separation be less than the time at which  $\theta_c$  is exceeded. Brouwer and Postema<sup>8</sup> have presented an approximate equation for determining the time required for separation. Assuming their optimal condition (conductivity of the terminating zone approaching that of the sample solution) and using a mobility difference of  $2 \cdot 10^{-4} \text{ cm}^2 \cdot \text{V}^{-1} \cdot \text{sec}^{-1}$ , an adapted sample length of 1 cm, and the data of Table I, their result may be written as

$$\tau_{\rm s}=0.0052965\,\frac{E}{S}.$$



Fig. 8. Steady-state temperature profiles based on constant electrical conductivity (plotted from results given in ref. 2). Comparison with Fig. 6 shows the degree to which results based on constant electrical conductivity underpredict the actual temperature distribution.  $\xi_t = 0.8$ ,  $\gamma = \infty$ 

From plots such as those in Figs. 1-3, one can determine for a given  $\xi_1$  the variation of  $\tau_c$  with S. Then, if one stipulates a value for the ratio  $\tau_s/\tau_c$  (thus assuring  $\tau_s < \tau_c$ ), one may determine the variation of  $\tau_s$  with S. Using this and the above equation, one can determine how E varies with S. Finally, using the definition of S one can determine the corresponding values of  $R_2$ , and hence also  $R_1$ , for the given  $\xi_1$ . Fig. 9



Fig. 9. Variation of internal column radius with electric field for  $\tau_s/\tau_c = 0.8$ . This figure clearly demonstrates the advantages of thick-walled columns.

shows the variation of  $R_1$  with E for various values of  $\xi_1$  for a value of  $\tau_s/\tau_c$  of 0.8. Again the advantage of thick-walled columns can be seen. For a given internal wall radius, the use of thicker-walled columns allows larger values of electric field strength to be used, thus increasing resolution. On the other hand, for a given value of electric field strength, thicker-walled columns allow the use of larger internal radii. This fact is particularly important for preparative electrophoresis.

## CONCLUSION

The present analysis, based on a linear variation of electrical conductivity with temperature and taking into account the effects of a finite wall thickness, is seen to be a major improvement over previous analyses, which predict parabolic temperature distributions. Previous analyses are seen to underpredict considerably the actual temperature variation in the isotachophoretic column, leading to the possibility of designs allowing temperature rises hazardous to living cells. It was shown that for higher resolution, operation at values of S greater than  $S_{max}$ . might be desirable, thus necessitating the solution to the transient problem presented here. It was also shown to be advantageous to use thick-walled columns both from the point of view of higher resolution and of increased production.

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# APPENDIX

COMPARISON OF "EXACT" SOLUTION OF STEADY-STATE TEMPERATURE DISTRI-BUTION ASSUMING CONSTANT THERMAL PROPERTIES WITH A STEADY-STATE PERTURBATION SOLUTION PERMITTING LINEAR TEMPERATURE DEPENDENCE OF THERMAL PROPERTIES

# (1) The constant-properties solution

The solution of the steady-state problem assuming constant thermal properties (density, specific heat, and thermal conductivity) has been given. For the special case where  $T_2 = T_{\infty}$  at  $r = R_2$  we have:

$$\theta_{1}^{s} = \frac{J_{0}(\beta\xi)}{\alpha \left[ J_{0}(\beta\xi_{1}) + \frac{k_{1}}{k_{2}} \beta\xi_{1} J_{1}(\beta\xi_{1}) \ln \xi_{1} \right]} - \frac{1}{\alpha}$$

from which the maximum temperature, at  $\xi = 0$ , is

$$\theta_{1_{\max}}^{s} = \frac{1}{\alpha} \left\{ \frac{1}{\left[ J_{0}\left(\beta\xi_{1}\right) + \frac{k_{1}}{k_{2}} \beta\xi_{1} J_{1}\left(\beta\xi_{1}\right) \ln \xi_{1} \right]} - 1 \right\}$$

For the purposes of obtaining a numerical comparison with the perturbation solution given below we have chosen the following values for the parameters involved:  $\xi_1 = 2/3$ ;  $\alpha = 7.75$ ;  $\theta_{j_{max}}^s = 0.1355$ ;  $k_1 = 13.9 \cdot 10^{-4} \text{ cal} \cdot \text{cm}^{-1} \cdot \text{sec}^{-1} \cdot ^{\circ}\text{K}^{-1}$ ;  $k_2 = 26.7 \cdot 10^{-4} \text{ cal} \cdot \text{cm}^{-1} \cdot \text{sec}^{-1} \cdot ^{\circ}\text{K}^{-1}$ .

The values of  $k_1$  and  $k_2$  are average values for the range of  $\theta^s$  considered, *i.e.*  $0 \leq \theta^s \leq \theta^s_{1_{max}}$ .

Solving for  $\beta = \sqrt{aS}$  then gives a value for the maximum permissible value of the dimensionless heat source strength S.

We find that  $S_{max.} = 0.480$ .

(2) Perturbation solution for the steady-state problem

The statement of the problem in terms of the dimensionless variables is:

$$\frac{1}{\xi} \frac{\mathrm{d}}{\mathrm{d}\xi} \left[ \xi \left( 1 + \beta_1 \, \theta_1^{\mathrm{s}} \right) \frac{\mathrm{d}\theta_1^{\mathrm{s}}}{\mathrm{d}\xi} \right] = -S \left( 1 + \alpha \theta_1^{\mathrm{s}} \right), \ 0 \leqslant \xi \leqslant \xi_1$$
$$\frac{1}{\xi} \frac{\mathrm{d}}{\mathrm{d}\xi} \left[ \xi \left( 1 + \beta_2 \, \theta_2^{\mathrm{s}} \right) \frac{\mathrm{d}\theta_2^{\mathrm{s}}}{\mathrm{d}\xi} \right] = 0, \ \xi_1 \leqslant \xi \leqslant 1$$

where now

$$k_{1} = k_{10} (1 + \beta_{1} \theta_{1}^{s})$$
  
$$k_{2} = k_{20} (1 + \beta_{2} \theta_{2}^{s})$$

and  $k_{10}$ ,  $k_{20}$  are the values of  $k_1$ ,  $k_2$  at  $\theta_{1,2}^s = 0$ . Boundary conditions are taken to be

$$\theta_2^s = 0 \text{ at } \xi = 1$$
  
$$\theta_1^s = \theta_2^s \text{ at } \xi = \xi_1$$
  
$$k_1 \frac{d\theta_1^s}{d\xi} = k_2 \frac{d\theta_2^s}{d\xi} \text{ at } \xi =$$

and the temperature at  $\xi = 0$  must be finite.

We proceed by assuming that  $\theta$  can be expanded in terms of a power series in the dimensionless source strength S.

$$\theta_1^{s}(\xi; S) \sim Sf_0(\xi) + S^2 f_1(\xi) + S^3 f_2(\xi) + \dots, \ 0 \leq \xi \leq \xi_1$$
  
$$\theta_2^{s}(\xi; S) \sim S\varphi_0(\xi) + S^2 \varphi_1(\xi) + S^3 \varphi_2(\xi) + \dots, \ \xi_1 \leq \xi \leq 1.$$

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The obvious procedural steps are not reproduced here; instead we shall quote the solution of  $\theta_1^s$  which we require for comparative purposes.

We obtain the following:

$$\theta_{1}^{s} = S\left[B_{1} - \frac{\xi^{2}}{4}\right] + S^{2}\left[\left(\beta_{1} - \alpha\right)\left(\frac{B_{1}\xi^{2}}{4} - \frac{\xi^{4}}{64}\right) - \frac{\beta_{1}\xi^{4}}{64} + D_{1}\right] + \dots, \\ 0 \leq \xi \leq \xi_{1}$$

where

$$B_{1} = \frac{\xi_{1}^{2}}{4} - \frac{k_{10}}{k_{20}} \frac{\xi_{1}^{2}}{2} \ln \xi_{1}$$

$$D_{1} = C \ln \xi_{1} - \frac{\beta_{2} A^{2}}{2} (\ln \xi_{1})^{2} - (\beta_{1} - \alpha) \left(\frac{B_{1} \xi_{1}^{2}}{4} - \frac{\xi_{1}^{4}}{64}\right) + \frac{\beta_{1} \xi_{1}^{4}}{64}$$

$$C = \frac{k_{10}}{k_{20}} \left[ (\beta_{1} - \alpha) \left(\frac{B_{1} \xi_{1}^{2}}{2} - \frac{\xi_{1}^{4}}{16}\right) - \frac{\beta_{1} \xi_{1}^{4}}{16} - \beta_{1} \left(\frac{B_{1} \xi_{1}^{2}}{2} - \frac{\xi_{1}^{4}}{8}\right) \right]$$

$$A = -\frac{k_{10}}{k_{20}} \frac{\xi_{1}^{2}}{2}$$

For  $\theta_{1_{max}}^{s}$  at  $\xi = 0$  we then have simply

$$\theta_{1_{max}}^{s} = SB_1 + S^2D_1 + \ldots$$

Numerical calculations assume the same values used in section 1 except that we now use:  $k_{20} = 26 \cdot 10^{-4} \text{cal} \cdot \text{cm}^{-1} \cdot \text{sec}^{-1} \cdot {}^{\circ}\text{K}^{-1}$ ;  $k_{10} = 13.19 \cdot 10^{-4} \text{cal} \cdot \text{cm}^{-1} \cdot \text{sec}^{-1} \cdot {}^{\circ}\text{K}^{-1}$ ;  $\beta_2 = 0.42$ ;  $\beta_1 = 0.808$ .

We now find a value for  $S_{max}$ .

 $S_{\text{max.}} = 0.576$  (variable properties).

This value may be compared directly with that obtained from the perturbation scheme by setting  $\beta_1 = \beta_2 = 0$  and replacing  $k_{10}$ ,  $k_{20}$  by the average values for  $k_1$ ,  $k_2$  used in section 1, in which case we get

 $S_{max.} = 0.561$  (constant properties).

Thus we see that within the perturbation steady state solution, taken to two terms, the effect of neglecting thermal property variations leads to a difference of some 2.5% in the calculation of  $S_{max}$ . Furthermore we note that the result obtained by assuming constant properties is conservative.

# (3) Comparison with the "exact" solution

It will be noted that the value of  $S_{max.} = 0.48$  given in section 1 is some 14.5% below that obtained from the two-term perturbation method also assuming constant properties. This, however, is a result of relatively slow convergence of the perturbation expansion. We have continued the expansion to three terms (assuming constant properties) and then obtain  $S_{max.} = 0.51$ , which is only 6% higher than the "exact" solution. We conclude therefore that the error in neglecting the thermal properties dependence on temperature is justified and is probably less than 3% in view of the comparisons made within the perturbation solutions. Moreover, the "exact" constant-properties solution gives a conservative answer and is therefore safe for design calculations.

# LIST OF SYMBOLS

Symbol	Meaning	Unit
c	Specific heat	cal · g <sup>-1</sup> · °K <sup>-1</sup>
Ε	Electric field	V/cm
h	Heat-transfer coeefficient	$cal \cdot cm^{-2} \cdot sec^{-1} \cdot {}^{\circ}K^{-1}$
J	Mechanical equivalent of heat	cal/J
$J_0$	Bessel function of the first kind of order zero	·
$J_1$	Bessel function of the first kind of order one	
K	Thermal diffusivity	cm²/sec
k	Thermal conductivity	$cal \cdot cm^{-1} \cdot sec^{-1} \cdot {}^{\circ}K^{-1}$
$Q_e$	Heat generation per unit volume	$cal \cdot cm^{-3} \cdot sec^{-1}$
r	Radial distance	cm
$R_1$	Internal radius of column	cm
$R_2$	External radius of column	cm
ร	Dimensionless heating rate	
Т	Temperature	°K
t	Time	sec
Y <sub>0</sub>	Bessel function of the second kind of order zero	
Y <sub>1</sub>	Bessel function of the second kind of order one	
α	Dimensionless coefficient of electrical conductivity	
γ	Dimensionless heat-transfer coefficient	
0	Dimensionless temperature	
ξ	Dimensionless radial distance	
$\xi_1$	Dimensionless wall-thickness parameter	
e	Mass density	g/cm <sup>3</sup>
σ	Electrical conductivity	$\Omega^{-1} \cdot \mathrm{cm}^{-1}$
τ	Dimensionless time	
$\tau_{c}$	Dimensionless time required to reach $\theta_{c}$	
$ au_{s}$	Dimensionless time required for separation	
Subscripts		
1	Refers to properties of the electrolyte solution.	
2	Refers to properties of the column wall.	
~	Refers to properties of the cooling bath	
	surrounding the isotachophoresis column.	
Superscripts		
S	Refers to steady-state conditions.	
t	Refers to transient conditions.	

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